

Drag in Cavitating Flow*

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The free streamline theory has been used for evaluation of the cavity drag of symmetrical wedges of arbitrary angle. The required conformal transformation is derived explicitly. This calculation is an extension of Riabouchinsky's theory of the cavity drag of a flat plate. As an approximation, the pressure distribution for a two-dimensional wedge is used to calculate the cavity drag of the corresponding cone of revolution. A comparison of the result of this approximation with experimental measurements made by Reichardt shows good agreement.

INTRODUCTION

THE class of two-dimensional motions of a perfect fluid in which the fluid boundaries consist in part of fixed walls and in part of surfaces of constant pressure was first considered by Helmholtz; general methods of attack on problems of this type have been given by Kirchhoff, Rayleigh, and others.¹ Of classical interest is the problem of a flat plate normal to the direction of flow in an infinite stream. The drag coefficient is found to be

$$C_D = \frac{2\pi}{\pi+4} = 0.880.$$

This drag coefficient is known to be much lower than the experimental value so that this free streamline theory of the wake cannot be accepted. A similar discrepancy is exhibited in the solution for a flat plate inclined at an angle α with the direction of flow of the infinite stream. In this case, the free streamline theory gives, for the force per unit area in the direction normal to the stream, the coefficient¹

$$C_L = 2\pi \sin\alpha \cos\alpha / (4 + \pi \sin\alpha).$$

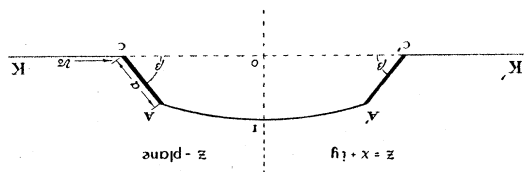


FIG. 1.

The circulation theory, which is found to be in agreement with experiment, at least for small α , gives on the other hand

$$C_L = 2\pi \sin\alpha.$$

The physical reason for the failure of the free streamline theory of the wake is clear: the free streamline theory considers the wake as consisting of "dead water," or fluid at rest, up to the bounding free streamlines; actually, the wake has vortex rows which produce a suction on the rear surface of the plate comparable to the maximum pressures on the forward surface.

While the free streamline theory of the wake is only of mathematical interest in applications to fluid flow of one phase, the theory should have some physical significance for flow of liquids. As the relative velocity of flow of a liquid over an immersed body is increased, the local pressure at some points on the surface of the body will fall to the vapor pressure of the liquid. The flow may then go over into a cavity wake. While the vapor cavity has its vortex system, the inertia of the vapor is negligible compared with that of the liquid, and the kinetic energy of the vortex system has only a trivial influence on the pressure within the wake. One would then expect that the free streamline theory of the wake would apply quantitatively to this two-phase situation.

The parameter which determines the general character of cavity flow is the cavitation param-

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¹ Cf. H. Lamb. *Hydrodynamics* (Dover Publications, New York, 1945), 6th ed. Chap. IV, §§73-78; U. Cisotti, *Idromeccanica Piana* (Tamburini, Milan, 1921-22), Part 2.

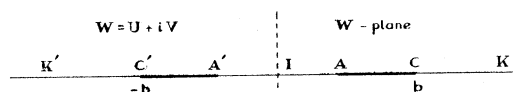


FIG. 2.

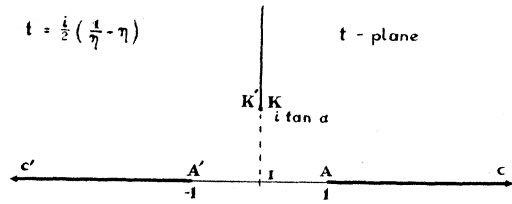


FIG. 5.

which represents the "physical" or z -plane ($z=x+iy$). Only the flow above the line of symmetry $KCC'K'$ is considered; the complete flow pattern is obtained merely by reflection in this line. Since all the results of the theory are independent of the magnitude of the velocity, there is no loss in generality in assigning the value unity to the velocity on the free streamline AA' ; the velocity at infinity is v_0 . The complex potential is $W=U+iV$ where U is the velocity potential and V is the stream function. The W -plane is shown in Fig. 2; the arbitrary constants in W have been fixed so that $V=0$ on the stream line $KCAIA'C'K'$ and $U=0$ at the mid-

point, I , of the free stream line. If u, v are x, y components of the fluid velocity, and if the convention is followed that

$$u = -\partial U / \partial x, \quad v = -\partial U / \partial y,$$

then

$$dW/dz = \zeta = -u + iv; \quad (2)$$

i.e., ζ is the reflection of the complex velocity in the y . The hodograph, or ζ -plane, is shown in Fig. 3.

The mathematical problem reduces to the determination of the unique one-to-one conformal transformation $W=W(\zeta)$ which maps the interior of the ζ -sector on the upper half W -plane so that corresponding boundary points coincide. The solution is completed by integration of the relation $\zeta=dW/dz$. The ζ -sector $ACC'A$ goes into a semicircle of unit radius in the η -plane (Fig. 4) by the transformation

$$\eta = \zeta^{\pi/2\beta}. \quad (3)$$

Further, the transformation

$$t = \frac{i}{2} \left(\frac{1}{\eta} - \eta \right) \quad (4)$$

takes the boundary of the semicircle in the η -plane into the real t axis; the interior of the semicircle goes into the upper half t -plane (Fig. 5). In particular, KK' for which $\eta = v_0^{\pi/2\beta}$ goes into $t = i \tan \alpha$ where the parameter α is determined by the relation

$$v_0^{\pi/2\beta} = (1 - \sin \alpha) / \cos \alpha. \quad (5)$$

Finally, the transformation

$$t = W \tan \alpha / (b^2 - W^2)^{1/2}; \quad W = bt / (\tan^2 \alpha + t^2)^{1/2} \quad (6)$$

takes the upper half t -plane into the upper half W -plane with the desired correlation of the boundary points. The parameter b fixes the cavity length CC' .

The drag coefficient, C_D , is expressed as follows:

$$C_D = \int_{AC} (p - p_1) dy / \frac{1}{2} \rho v_0^2 \int_{AC} dy, \quad (7)$$

where p is the fluid pressure over the upstream face of the wedge and p_1 is the pressure in the

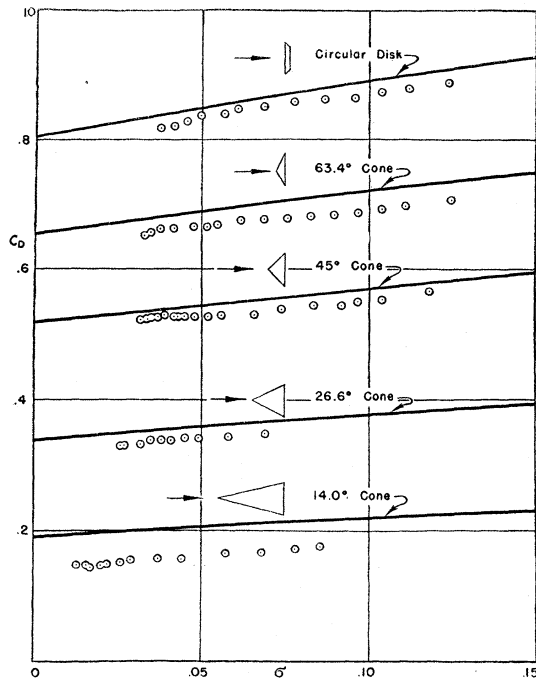


FIG. 6. The drag coefficient C_D as ordinate is plotted as a function of the cavitation parameter σ as abscissa. The lines are calculated by the approximation indicated in the text, and the circled points are the measured values of Reichardt. The angles indicated are the interior half angles of the symmetric cones of revolution.

cavity. Also, from Bernoulli's equation,

$$\begin{aligned} p - p_1 &= (\rho/2)(1 - v^2) \\ &= (\rho/2)(1 - |\zeta|^2), \end{aligned} \quad (8)$$

where ζ is to be evaluated along AC . Thus,

$$C_D = (1/v_0^2) \cdot \left(\int_{AC} (1 - |\zeta|^2) dy / \int_{AC} dy \right). \quad (9)$$

APPROXIMATE CAVITY DRAG FOR CONES OF REVOLUTION

Thus far, there has not been available a theoretical evaluation of the cavity drag of bodies of revolution. The mathematical complexity of such problems is very great; yet it is for bodies of this type for which there is the most direct experimental interest. A calculation of the cavity drag for cones of revolution has been made by the authors under the assumption that the pressure distribution ($p - p_1$) over the

cone is the same as that found from the two-dimensional flow about a wedge of the same internal angle. The accuracy of such an approximation is a question which must be determined by experiment, and if the approximation is shown to be reasonably good, support is given to general application of this procedure for the determination of cavity drag for bodies of revolution.

The only experimental data available to the writers bearing on this question are those of Reichardt. A comparison of the results of the present approximation and Reichardt's data is given in Fig. 6. The agreement is quite satisfactory except for the case of the cone of half-angle $\beta = 14.0^\circ$. An explanation of this particular discrepancy was not obvious to the writers in view of the good agreement elsewhere. It would appear most desirable to have further experimentation in this field over the widest possible range of variation of cavitation parameter.